

Elasticity required when studying plasticity

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Constitutive description on elasticity

Elastic constitutive law:

$$\mathbb{E}\varepsilon = \sigma \text{ (elastic stiffness } \mathbb{E} = 200 \text{ [GPa])}$$

Elastic constitutive law:

$$\mathbb{E}_{ijkl}\varepsilon_{kl} = \sigma_{ij}$$

Apply it to **F**ORTRAN, (or **P**ython or **E**xcel)

-Exercise 1. $[3 \times 1] = [3 \times 3] [3 \times 1]$

-Exercise 2. $[n \times 1] = [n \times n] [n \times 1]$

- (hint): use $[A]_i = [B]_{ij}[C]_j$

Kronecker delta may appear in formula

$$\begin{aligned} a &= \varepsilon_{kl} \delta_{kl} = \sum_k \sum_l \varepsilon_{kl} \delta_{kl} = \sum_l \varepsilon_{1l} \delta_{1l} + \sum_l \varepsilon_{2l} \delta_{2l} + \sum_l \varepsilon_{3l} \delta_{3l} \\ &= \varepsilon_{11} \delta_{11} + \varepsilon_{22} \delta_{22} + \varepsilon_{33} \delta_{33} = \sum_k \varepsilon_{kk} = \sum_i \varepsilon_{ii} = \sum_l \varepsilon_{ll} \\ &= \varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33} \end{aligned}$$

Linear isotropic elasticity

Elastic constitutive law (Hooke's law):

$$\mathbb{E}_{ijkl}\varepsilon_{kl} = \sigma_{ij} \quad (\text{linear elasticity})$$

$$\mathbb{E}_{ijkl} = \lambda\delta_{ij}\delta_{kl} + \mu(\delta_{ik}\delta_{jl} + \delta_{il}\delta_{jk}) \quad (\text{isotropic elasticity; two constants } \lambda, \mu)$$

Replacing $\mathbb{E}_{ijkl}$ to the Hooke's law

$$\begin{aligned}\sigma_{ij} &= \mathbb{E}_{ijkl}\varepsilon_{kl} = \lambda\delta_{ij}\delta_{kl}\varepsilon_{kl} + \mu(\delta_{ik}\delta_{jl} + \delta_{il}\delta_{jk})\varepsilon_{kl} \\ &= \lambda\delta_{ij}\varepsilon_{kk} + \mu(\delta_{ik}\delta_{jl}\varepsilon_{kl} + \delta_{il}\delta_{jk}\varepsilon_{kl}) = \lambda\delta_{ij}\varepsilon_{kk} + \mu(\delta_{ik}\varepsilon_{kj} + \delta_{il}\varepsilon_{jl}) \\ &= \lambda\delta_{ij}\varepsilon_{kk} + \mu(\varepsilon_{ij} + \varepsilon_{ji}) = \lambda\delta_{ij}\varepsilon_{kk} + 2\mu\varepsilon_{ij}\end{aligned}$$

Demonstration with Excel

Examples

- In order for a material (with $\lambda = 115.384$ GPa, $\mu = 76.923$ GPa) to exhibit below elastic strain, what stress should be given?

$$\varepsilon = \begin{bmatrix} 0.002 & 0 & 0 \\ 0 & -0.0006 & 0 \\ 0 & 0 & -0.0006 \end{bmatrix}$$

- Hint: use “ $\sigma_{ij} = \lambda \delta_{ij} \varepsilon_{kk} + 2\mu \varepsilon_{ij}$ ”

$$\begin{bmatrix} 200.000 & 0.000 & 0.000 \\ 0.000 & 0.000 & 0.000 \\ 0.000 & 0.000 & 0.000 \end{bmatrix}$$

Linear **isotropic** elasticity (Young, Poisson)

Elastic constitutive law (Hooke's law):

$$\varepsilon_{ij} = \frac{1}{E} [\sigma_{ij} - \nu(\sigma_{kk}\delta_{ij} - \sigma_{ij})]$$

- If you apply below stress to a material (with $E = 200$ GPa, $\nu = 0.3$), in what strain tensor will the material exhibit?

$$\sigma = \begin{bmatrix} 200 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \quad \text{the unit of stress is MPa}$$

$$\begin{bmatrix} 0.00100 & 0.00000 & 0.00000 \\ 0.00000 & -0.00030 & 0.00000 \\ 0.00000 & 0.00000 & -0.00030 \end{bmatrix}$$

Notice that a material with $E = 200$ GPa, $\nu = 0.3$ behaves equivalently with a material with $\lambda = 115.384$ GPa, $\mu = 76.923$ GPa

Demonstration with Excel

Symmetries; why only **two** parameters?

- $\sigma_{ij} = \sigma_{ji}$ gives $\mathbb{E}_{ijkl} = \mathbb{E}_{jikl}$ thus, the required number of elastic constants reduces from $3 \times 3 \times 3 \times 3$ to $6 \times 3 \times 3$.
- Similarly, $\varepsilon_{ij} = \varepsilon_{ji}$ gives $\mathbb{E}_{ijkl} = \mathbb{E}_{ijlk}$ so that we have the required number of constants $6 \times 6 = 36$

The required number of constants can be further reduced. Consider the elastic energy:

$$\begin{aligned}\phi &= \int \sigma_{ij} d\varepsilon_{ij} \\ \sigma_{ij} &= \frac{\partial \phi}{\partial \varepsilon_{ij}} = \mathbb{E}_{ijkl} \varepsilon_{kl}\end{aligned}$$

If we apply partial derivative once again, we have

$$\frac{\partial^2 \phi}{\partial \varepsilon_{mn} \partial \varepsilon_{ij}} = \frac{\partial}{\partial \varepsilon_{mn}} (\mathbb{E}_{ijkl} \varepsilon_{kl}) \text{ since } \mathbb{E} \text{ is 'constant', we have}$$

$$\frac{\partial^2 \phi}{\partial \varepsilon_{mn} \partial \varepsilon_{ij}} = \mathbb{E}_{ijkl} \left(\frac{\partial \varepsilon_{kl}}{\partial \varepsilon_{mn}} \right) = \mathbb{E}_{ijkl} \delta_{km} \delta_{ln} = \mathbb{E}_{ijmn}$$

Symmetries; why only **two** parameters?

- $\phi = \int \sigma_{ij} d\varepsilon_{ij}$
- $\sigma_{ij} = \frac{\partial \phi}{\partial \varepsilon_{ij}} = \mathbb{E}_{ijkl} \varepsilon_{kl}$
- If we apply partial derivative once again, we have
- $\frac{\partial^2 \phi}{\partial \varepsilon_{mn} \partial \varepsilon_{ij}} = \frac{\partial}{\partial \varepsilon_{mn}} (\mathbb{E}_{ijkl} \varepsilon_{kl})$ since $\mathbb{E}$ is 'constant', we have
- $\frac{\partial^2 \phi}{\partial \varepsilon_{mn} \partial \varepsilon_{ij}} = \mathbb{E}_{ijkl} \left(\frac{\partial \varepsilon_{kl}}{\partial \varepsilon_{mn}} \right) = \mathbb{E}_{ijkl} \delta_{km} \delta_{ln} = \mathbb{E}_{ijmn}$
- We could do the 2nd order derivative in a different way (say, instead of $\frac{\partial^2 \phi}{\partial \varepsilon_{mn} \partial \varepsilon_{ij}}$ we could have done $\frac{\partial^2 \phi}{\partial \varepsilon_{ij} \partial \varepsilon_{mn}} = \frac{\partial}{\partial \varepsilon_{ij}} \left(\frac{\partial \phi}{\partial \varepsilon_{mn}} \right) = \frac{\partial}{\partial \varepsilon_{ij}} (\mathbb{E}_{mni j}) = \mathbb{E}_{mni j}$
- The two cases (regardless of the order of derivative) should give equivalent result so that
- $\mathbb{E}_{ijmn} = \mathbb{E}_{mni j}$
- This summarizes our finding on the symmetries in elastic tensor:

Reduction to Voigt notation

- $$\sigma_{21} = \mathbb{E}_{2111}\varepsilon_{11} + \mathbb{E}_{2112}\varepsilon_{12} + \mathbb{E}_{2113}\varepsilon_{13} + \mathbb{E}_{2121}\varepsilon_{21} + \mathbb{E}_{2122}\varepsilon_{22} + \mathbb{E}_{2123}\varepsilon_{23} + \mathbb{E}_{2131}\varepsilon_{31} + \mathbb{E}_{2132}\varepsilon_{32} + \mathbb{E}_{2133}\varepsilon_{33}$$

- $$\sigma_{21} = \begin{bmatrix} \mathbb{E}_{2111} \\ \mathbb{E}_{2112} \\ \mathbb{E}_{2113} \\ \mathbb{E}_{2121} \\ \mathbb{E}_{2122} \\ \mathbb{E}_{2123} \\ \mathbb{E}_{2131} \\ \mathbb{E}_{2132} \\ \mathbb{E}_{2133} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{12} \\ \varepsilon_{13} \\ \varepsilon_{21} \\ \varepsilon_{22} \\ \varepsilon_{23} \\ \varepsilon_{31} \\ \varepsilon_{32} \\ \varepsilon_{33} \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{2111} \\ 2\mathbb{E}_{2112} \\ 2\mathbb{E}_{2113} \\ - \\ \mathbb{E}_{2122} \\ 2\mathbb{E}_{2123} \\ - \\ - \\ \mathbb{E}_{2133} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{12} \\ \varepsilon_{13} \\ - \\ \varepsilon_{22} \\ \varepsilon_{23} \\ - \\ - \\ \varepsilon_{33} \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{2111} \\ \mathbb{E}_{2122} \\ \mathbb{E}_{2133} \\ 2\mathbb{E}_{2123} \\ 2\mathbb{E}_{2113} \\ 2\mathbb{E}_{2112} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{23} \\ \varepsilon_{13} \\ \varepsilon_{12} \end{bmatrix}$$

- $$or = \begin{bmatrix} \mathbb{E}_{2111} \\ \mathbb{E}_{2122} \\ \mathbb{E}_{2133} \\ \mathbb{E}_{2123} \\ \mathbb{E}_{2113} \\ \mathbb{E}_{2112} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{23} \\ \gamma_{13} \\ \gamma_{12} \end{bmatrix} \text{ with } \gamma_{12} = 2\varepsilon_{12} \text{ and so forth}$$

$$\sigma_{21} = \begin{bmatrix} \mathbb{E}_{21,1} \\ \mathbb{E}_{21,2} \\ \mathbb{E}_{21,3} \\ \mathbb{E}_{21,4} \\ \mathbb{E}_{21,5} \\ \mathbb{E}_{21,6} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \gamma_4 \\ \gamma_5 \\ \gamma_6 \end{bmatrix}$$

with (1,1) → (1), (2,2) → (2), (3,3) → (3)
 (2,3) → (4), (1,3) → (5), (1,2) → (6)

Reduction to Voigt notation

$$\sigma_{21} = \begin{bmatrix} \mathbb{E}_{21,1} \\ \mathbb{E}_{21,2} \\ \mathbb{E}_{21,3} \\ \mathbb{E}_{21,4} \\ \mathbb{E}_{21,5} \\ \mathbb{E}_{21,6} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \gamma_4 \\ \gamma_5 \\ \gamma_6 \end{bmatrix}$$

with $(1,1) \rightarrow (1)$, $(2,2) \rightarrow (2)$, $(3,3) \rightarrow (3)$
 $(2,3) \rightarrow (4)$, $(1,3) \rightarrow (5)$, $(1,2) \rightarrow (6)$

$$\sigma_{21} = \begin{bmatrix} \mathbb{E}_{21,1} \\ \mathbb{E}_{21,2} \\ \mathbb{E}_{21,3} \\ \mathbb{E}_{21,4} \\ \mathbb{E}_{21,5} \\ \mathbb{E}_{21,6} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{bmatrix}$$

with $(1,2) \rightarrow (2,1) \rightarrow (6)$

$$\sigma_6 = \begin{bmatrix} \mathbb{E}_{6,1} \\ \mathbb{E}_{6,2} \\ \mathbb{E}_{6,3} \\ \mathbb{E}_{6,4} \\ \mathbb{E}_{6,5} \\ \mathbb{E}_{6,6} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{bmatrix}$$

with $(1,2) \rightarrow (2,1) \rightarrow (6)$

Reduction to Voigt notation

$$\sigma_{ij} = \mathbb{E}_{ijkl} \varepsilon_{kl}$$

$$\sigma_i = \mathbb{E}_{ij} \varepsilon_j$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{1111} & \mathbb{E}_{1122} & \mathbb{E}_{1133} & \mathbb{E}_{1123} & \mathbb{E}_{1113} & \mathbb{E}_{1112} \\ \mathbb{E}_{2211} & \mathbb{E}_{2222} & \mathbb{E}_{2233} & \mathbb{E}_{2223} & \mathbb{E}_{2213} & \mathbb{E}_{2212} \\ \mathbb{E}_{3311} & \mathbb{E}_{3322} & \mathbb{E}_{3333} & \mathbb{E}_{3323} & \mathbb{E}_{3313} & \mathbb{E}_{3312} \\ \mathbb{E}_{2311} & \mathbb{E}_{2322} & \mathbb{E}_{2333} & \mathbb{E}_{2323} & \mathbb{E}_{2313} & \mathbb{E}_{2312} \\ \mathbb{E}_{1311} & \mathbb{E}_{1322} & \mathbb{E}_{1333} & \mathbb{E}_{1323} & \mathbb{E}_{1313} & \mathbb{E}_{1312} \\ \mathbb{E}_{1211} & \mathbb{E}_{1222} & \mathbb{E}_{1233} & \mathbb{E}_{1223} & \mathbb{E}_{1213} & \mathbb{E}_{1212} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ 2\varepsilon_{23} \\ 2\varepsilon_{13} \\ 2\varepsilon_{12} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{11} & \mathbb{E}_{12} & \mathbb{E}_{13} & \mathbb{E}_{14} & \mathbb{E}_{15} & \mathbb{E}_{16} \\ \mathbb{E}_{21} & \mathbb{E}_{22} & \mathbb{E}_{23} & \mathbb{E}_{24} & \mathbb{E}_{25} & \mathbb{E}_{26} \\ \mathbb{E}_{31} & \mathbb{E}_{32} & \mathbb{E}_{33} & \mathbb{E}_{34} & \mathbb{E}_{35} & \mathbb{E}_{36} \\ \mathbb{E}_{41} & \mathbb{E}_{42} & \mathbb{E}_{43} & \mathbb{E}_{44} & \mathbb{E}_{45} & \mathbb{E}_{46} \\ \mathbb{E}_{51} & \mathbb{E}_{52} & \mathbb{E}_{53} & \mathbb{E}_{54} & \mathbb{E}_{55} & \mathbb{E}_{56} \\ \mathbb{E}_{61} & \mathbb{E}_{62} & \mathbb{E}_{63} & \mathbb{E}_{64} & \mathbb{E}_{65} & \mathbb{E}_{66} \end{bmatrix} \begin{bmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \\ \epsilon_4 \\ \epsilon_5 \\ \epsilon_6 \end{bmatrix}$$

How many constants are required?

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{1111} & \mathbb{E}_{1122} & \mathbb{E}_{1133} & \mathbb{E}_{1123} & \mathbb{E}_{1113} & \mathbb{E}_{1112} \\ \mathbb{E}_{2211} & \mathbb{E}_{2222} & \mathbb{E}_{2233} & \mathbb{E}_{2223} & \mathbb{E}_{2213} & \mathbb{E}_{2212} \\ \mathbb{E}_{3311} & \mathbb{E}_{3322} & \mathbb{E}_{3333} & \mathbb{E}_{3323} & \mathbb{E}_{3313} & \mathbb{E}_{3312} \\ \mathbb{E}_{2311} & \mathbb{E}_{2322} & \mathbb{E}_{2333} & \mathbb{E}_{2323} & \mathbb{E}_{2313} & \mathbb{E}_{2312} \\ \mathbb{E}_{1311} & \mathbb{E}_{1322} & \mathbb{E}_{1333} & \mathbb{E}_{1323} & \mathbb{E}_{1313} & \mathbb{E}_{1312} \\ \mathbb{E}_{1211} & \mathbb{E}_{1222} & \mathbb{E}_{1233} & \mathbb{E}_{1223} & \mathbb{E}_{1213} & \mathbb{E}_{1212} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ 2\varepsilon_{23} \\ 2\varepsilon_{13} \\ 2\varepsilon_{12} \end{bmatrix}$$

How many constants do we need?

If the coordinate system happens to give strain and stress all principal values:

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \end{bmatrix} = \begin{bmatrix} \mathbb{E}_{1111} & \mathbb{E}_{1122} & \mathbb{E}_{1133} \\ & \mathbb{E}_{2222} & \mathbb{E}_{2223} \\ & & \mathbb{E}_{3333} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \end{bmatrix}$$

example

- Fe(1-0.025)-Al(0.025) alloy의 탄성 계수는 다음과 같이 주어진다.
- $E_{11} = 270.71$, $E_{12} = 128.03$, $E_{44} = 108.77$
- Fe-Al alloy는 Body-centered cubic 결정 구조를 가지고, 결정 대칭성에 의해 다음과 같은 탄성 거동을 한다.

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix} = \begin{bmatrix} E_{11} & E_{12} & E_{13} & 0 & 0 & 0 \\ E_{21} & E_{22} & E_{23} & 0 & 0 & 0 \\ E_{31} & E_{32} & E_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & E_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & E_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & E_{66} \end{bmatrix} \begin{bmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \\ \epsilon_4 \\ \epsilon_5 \\ \epsilon_6 \end{bmatrix}$$

- 뿐만 아니라, cubic 결정구조의 대칭성으로 인해 $E_{11} = E_{22} = E_{33}$, $E_{44} = E_{55} = E_{66}$, $E_{12} = E_{13} = E_{23}$

Example

- Fe(1-0.025)-Al(0.025) alloy의 단결정에 다음과 같은 탄성 변형률이 나타나기 위해 필요한 응력 상태는?

$$\begin{bmatrix} 0.0001 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Voigt notation

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1123} & C_{1113} & C_{1112} \\ & C_{2222} & C_{2233} & C_{2223} & C_{2213} & C_{2212} \\ & & C_{3333} & C_{3323} & C_{3313} & C_{3312} \\ & & & C_{2323} & C_{2313} & C_{2312} \\ & \text{symm} & & & C_{1313} & C_{1312} \\ & & & & & C_{1212} \end{bmatrix} \begin{bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{33} \\ 2\epsilon_{23} \\ 2\epsilon_{13} \\ 2\epsilon_{12} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & C_{15} & C_{16} \\ & C_{22} & C_{23} & C_{24} & C_{25} & C_{26} \\ & & C_{33} & C_{34} & C_{35} & C_{36} \\ & & & C_{44} & C_{45} & C_{46} \\ & \text{symm} & & & C_{55} & C_{56} \\ & & & & & C_{66} \end{bmatrix} \begin{bmatrix} \epsilon_1 \\ \epsilon_2 \\ \epsilon_3 \\ \epsilon_4 \\ \epsilon_5 \\ \epsilon_6 \end{bmatrix}$$

Cartesian <-> Voigt (cheat sheet)

```
1c stress, stiffness
2 SUBROUTINE VOIGT(T1,T2,C2,C4,IOPT)
3 IMPLICIT NONE
4 REAL*8 T1(6),T2(3,3),C2(6,6),C4(3,3,3,3)
5 INTEGER IJV(6,2),I,J,IOPT,I1,I2,J1,J2,N,M
6 DATA ((IJV(N,M),M=1,2),N=1,6)/1,1,2,2,3,3,1,2,1,3,2,3/
7
8 IF(IOPT.EQ.1) THEN
9   DO I=1,6
10    I1=IJV(I,1)
11    I2=IJV(I,2)
12    T2(I1,I2)=T1(I)
13    T2(I2,I1)=T1(I)
14  ENDDO
15 ENDIF
16c
17 IF(IOPT.EQ.2) THEN
18   DO I=1,6
19    I1=IJV(I,1)
20    I2=IJV(I,2)
21    T1(I)=T2(I1,I2)
22  ENDDO
23 ENDIF
24c
25 IF (IOPT.EQ.3) THEN
26   DO I=1,6
27    I1=IJV(I,1)
28    I2=IJV(I,2)
29    DO J=1,6
30     J1=IJV(J,1)
31     J2=IJV(J,2)
32     C4(I1,I2,J1,J2)=C2(I,J)
33     C4(I2,I1,J1,J2)=C2(I,J)
34     C4(I1,I2,J2,J1)=C2(I,J)
35     C4(I2,I1,J2,J1)=C2(I,J)
36   ENDDO
37 ENDDO
38 ENDIF
```

```
39c
40 IF(IOPT.EQ.4) THEN
41   DO I=1,6
42    I1=IJV(I,1)
43    I2=IJV(I,2)
44    DO J=1,6
45     J1=IJV(J,1)
46     J2=IJV(J,2)
47     C2(I,J)=C4(I1,I2,J1,J2)
48   ENDDO
49 ENDDO
50 ENDIF
51c
52 RETURN
53 END
```

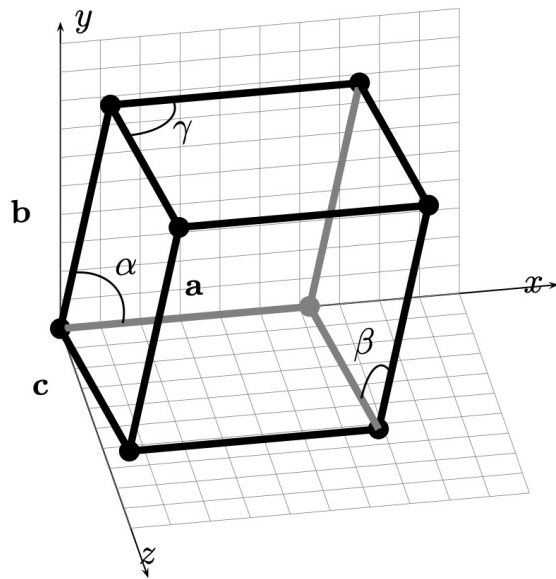
Convert from Cartesian to Voigt

- $\mathbb{E}_{11}^{(\text{voigt})} = \mathbb{E}_{1111}^{(\text{cartesian})}$, $\mathbb{E}_{23}^{(\text{voigt})} = \mathbb{E}_{2233}^{(\text{cartesian})}$, $\mathbb{E}_{41}^{(\text{voigt})} = \mathbb{E}_{2311}^{(\text{cartesian})}$
- Material anisotropy
- Symmetry can be represented by an orthogonal second order tensor,
- $\mathbf{Q} = Q_{ij} \mathbf{e}_i \otimes \mathbf{e}_j$, such that $\mathbf{Q}^{-1} = \mathbf{Q}^T$
- The **invariance** of the stiffness tensor under these transformations (due to symmetry) is:
- $\mathbb{E}^{(\text{new})} = \mathbf{Q} \cdot \mathbf{Q} \cdot \mathbb{E}^{(\text{old})} \cdot \mathbf{Q}^T \cdot \mathbf{Q}^T$ due to symmetry the resulting tensor should be equivalent with the original one: $\mathbb{E}^{(\text{new})} \equiv \mathbb{E}^{(\text{old})}$

Convert from Cartesian to Voigt

- $\mathbb{E}_{11}^{(\text{voigt})} = \mathbb{E}_{1111}^{(\text{cartesian})}$, $\mathbb{E}_{23}^{(\text{voigt})} = \mathbb{E}_{2233}^{(\text{cartesian})}$, $\mathbb{E}_{41}^{(\text{voigt})} = \mathbb{E}_{2311}^{(\text{cartesian})}$
- Material anisotropy
- Symmetry can be represented by an orthogonal second order tensor,
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Triclinic (no symmetry)



Triclinic: no symmetry planes, fully anisotropic.

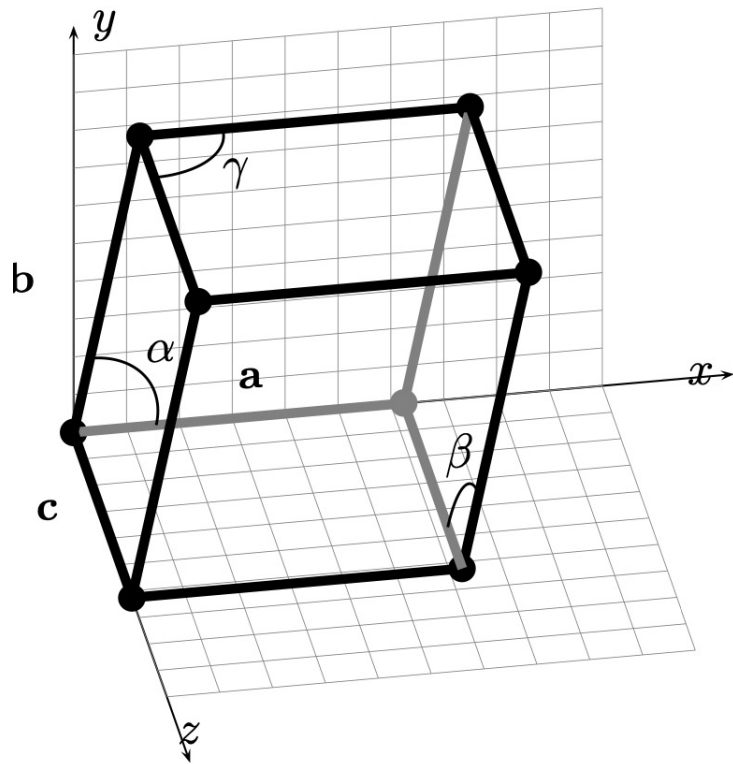
$\alpha, \beta, \gamma < 90$

Number of independent coefficients: 21

Symmetry transformation: None

$$C = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1123} & C_{1113} & C_{1112} \\ & C_{2222} & C_{2233} & C_{2223} & C_{2213} & C_{2212} \\ & & C_{3333} & C_{3323} & C_{3313} & C_{3312} \\ & & & C_{2323} & C_{2313} & C_{2312} \\ & \text{symm} & & & C_{1313} & C_{1312} \\ & & & & & C_{1212} \end{bmatrix}$$

monoclinic (one symmetry plane)



Monoclinic: one symmetry plane (xy).

$a \neq b \neq c$, $\beta = \gamma = 90$, $\alpha < 90$

Number of independent coefficients: 13

Symmetry transformation: reflection about z -axis

$$Q = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & -1 \end{bmatrix}$$

$$C = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 0 & 0 & C_{1112} \\ & C_{2222} & C_{2233} & 0 & 0 & C_{2212} \\ & & C_{3333} & 0 & 0 & C_{3312} \\ & & & C_{2323} & C_{2313} & 0 \\ & & & & C_{1313} & 0 \\ & & & & & C_{1212} \end{bmatrix}$$

symm

monoclinic (one symmetry plane)

- For the case of Monoclinic:

- $$\mathbf{Q} = Q_{ij} \mathbf{e}_i \otimes \mathbf{e}_j = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & -1 \end{bmatrix}$$

- Let's take a look at the invariance due to symmetry

- $\mathbb{E}^{(new)} = \mathbf{Q} \cdot \mathbf{Q} \cdot \mathbb{E}^{(old)} \cdot \mathbf{Q}^T \cdot \mathbf{Q}^T$ due to symmetry the resulting tensor should be equivalent with the original one: $\mathbb{E}^{(new)} \equiv \mathbb{E}^{(old)}$

- In its matrix form:

- $$\mathbb{E}_{ijkl}^{(new)} = Q_{im} Q_{jn} Q_{ko} Q_{lp} \mathbb{E}_{mnop}^{(old)}$$

- Ex: $\mathbb{E}_{11}^{(voigt)} = \mathbb{E}_{1111} = Q_{1m} Q_{1n} Q_{1o} Q_{1p} \mathbb{E}_{mnop}^{(old)}$

- If you look at the matrix form of symmetry operator $\mathbf{Q}$ in the above, only diagonal components are non-zero. Therefore, $Q_{ij} = 0$ if $i \neq j$.

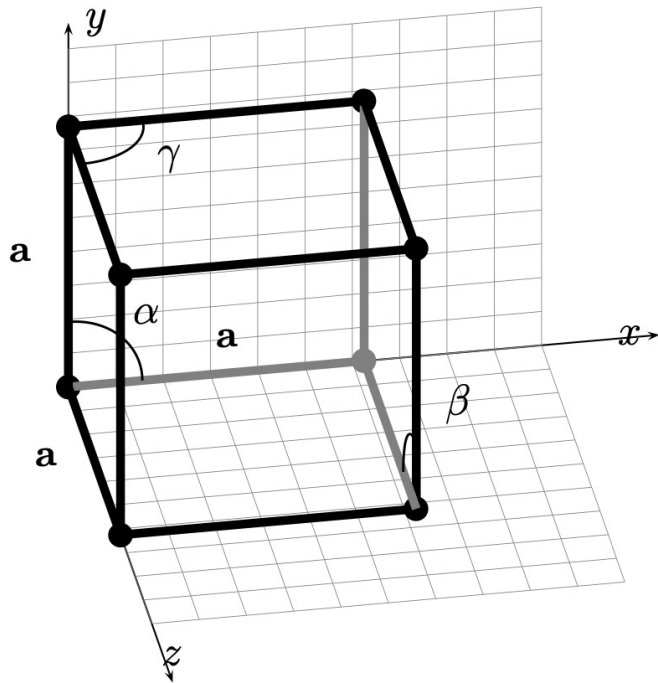
- $\mathbb{E}_{11}^{(voigt)} = \mathbb{E}_{1111} = Q_{11} Q_{11} Q_{11} Q_{11} \mathbb{E}_{1111}^{(old)} = \mathbb{E}_{1111}^{(old)}$

- Therefore, $\mathbb{E}_{11}^{(voigt)} = \mathbb{E}_{1111}$

- Ex: $\mathbb{E}_{14}^{(voigt)} = \mathbb{E}_{1123} = Q_{1m} Q_{1n} Q_{2o} Q_{3p} \mathbb{E}_{mnop}^{(old)} = Q_{11} Q_{11} Q_{22} Q_{33} \mathbb{E}_{1123}^{(old)} = 1 \times 1 \times 1 \times (-1) \times \mathbb{E}_{1123}^{(old)} = -\mathbb{E}_{1123}^{(old)}$

- Therefore, in order to satisfy $\mathbb{E}_{1123} = -\mathbb{E}_{1123}$, $\mathbb{E}_{1123}$ should be zero.

Cubic



Cubic: three mutually orthogonal planes of reflection symmetry plus 90° rotation symmetry with respect to those planes. $a = b = c$, $\alpha = \beta = \gamma = 90$
 Number of independent coefficients: 3

Symmetry transformations: reflections and 90° rotations about all three orthogonal planes

$$Q = \begin{bmatrix} -1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad Q = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad Q = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & -1 \end{bmatrix}$$

$$Q = \begin{bmatrix} 0 & 1 & 0 \\ -1 & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad Q = \begin{bmatrix} 0 & 0 & 1 \\ 0 & 1 & 0 \\ -1 & 0 & 0 \end{bmatrix} \quad Q = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & -1 & 0 \end{bmatrix}$$

$$C = \begin{bmatrix} C_{1111} & C_{1122} & C_{1122} & 0 & 0 & 0 \\ & C_{1111} & C_{1122} & 0 & 0 & 0 \\ & & C_{1111} & 0 & 0 & 0 \\ & & & C_{1212} & 0 & 0 \\ & \text{symm} & & & C_{1212} & 0 \\ & & & & & C_{1212} \end{bmatrix}$$